

Effects of variations in magnetic Reynolds number on magnetic field distribution in electrically conducting fluid under magnetohydrodynamic natural convection

Mohsen Pirmohammadi

Department of Mechanical Engineering, Pardis Branch, Islamic Azad University, Pardis, Iran

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ABSTRACT

This study numerically investigated the effects of variations in magnetic Reynolds number on magnetic field distribution in an enclosure under natural convection heat transfer. The investigated geometry was a two-dimensional enclosure with a hot left wall, a cold right wall, and adiabatic top and bottom walls. The fluid was molten sodium with $Pr=0.01$. Natural convection heat transfer at a Rayleigh number of 105 and magnetic Reynolds numbers (Rem) of 10–1, 10–3, and 10–5 was considered in the analysis. The governing equations adopted were continuum, momentum, energy, and magnetic induction equations, which were solved concurrently using the finite volume method. For the coupling of velocity and pressure, the revised semi-implicit method for pressure linked equations algorithm was employed. Results showed that under a high magnetic Reynolds number, the non-dimensional magnetic fields in the X and Y directions were approximately constant because the diffusion of the magnetic potential was greater than the advection of such potential. As the magnetic Reynolds number increased, however, the magnetic field in the enclosure reached a magnitude unequal to that of the applied magnetic field, exhibited inconstancy, and increased to a value that deviated from 1. Thus, the $Rem=10-1$ under a non-dimensional magnetic field increased from 0.09 to 6.6 in the X direction and from -1.164 to 4.05 in Y the direction.

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1. Introduction

The free convection heat transfer of electrically conducting fluids in enclosures has been the subject of a considerable number of theoretical, experimental, and numerical investigations because of its importance in many technological applications, such as use in liquid metal blankets for reactors. Another important application of the aforementioned process is crystal growth in the industrial production of semiconductors. This particular application has been explored in a number of works. Oreper and Szekely [1], for example, showed that a magnetic field suppresses natural convection currents and that magnetic field

strength is a critical factor for crystal formation. Hadid et al. [2] probed into the effects of a strong vertical magnetic field on convection and segregation in vertical Bridgeman crystal growth. Bessaih et al. [3] numerically examined the effects of the electric conductivity of walls and the direction of a magnetic field on gallium flow. The authors found a considerable reduction in convection intensity as the magnetic field increases.

Ciofalo and Cricchio [4] considered the MHD natural convection of liquid Pb–17Li in a cubical cavity with a uniform volumetric heat source and an induced current that significantly stretches to the walls normal to the applied magnetic field flux. Piazza and Ciofalo [5, 6] developed a simple wall boundary

condition to solve electromagnetism in a cavity without allocating real grid points to electrically conducting walls. Pirmohammadi et al. [7] numerically studied natural convection flow in the presence of a magnetic field in a tilted enclosure heated from below and filled with liquid gallium. The authors found that at a given inclination angle, convection heat transfer decreases when magnetic field intensity increases. Pirmohammadi et al. [8] studied the effects of a magnetic field on buoyancy-driven convection in a differentially heated square enclosure. The researchers showed that the heat transfer mechanisms and flow characteristics in the enclosure depend strongly on both the strength of the magnetic field and the Rayleigh number. They concluded that a magnetic field considerably decreases the average Nusselt number.

An extensive study of mixed convection was conducted by Selimefendigil et al. [11] for various geometries. Heidary et al. [12] numerically investigated the laminar flow and convective heat transfer of Cu-water nanofluid in a channel subjected to a uniform magnetic field. The authors revealed that the thickness of the thermal boundary layer increases with the addition of nanoparticles to pure fluid. Applying a magnetic field, however, thins the boundary layer because of the increase in velocity gradient near walls. Seth et al. [13] delved into the combined free and forced convection Couette–Hartmann flow of a viscous, incompressible, and electrically conducting fluid. The fluid was placed in a rotating channel with arbitrary conducting walls, and flow was examined under the presence of the Hall current. Makinde et al. [14] investigated the steady flow and heat transfer of an electrically conducting fluid with variable viscosity and electrical conductivity between two parallel plates in the presence of a transverse magnetic field.

In most of the above-mentioned studies, mathematical and numerical methods were intended to determine the influence of any combination of Grashof, Reynolds, Prandtl, and Hartmann numbers on flow and heat transfer. A technique common to these studies was the simplification of the magnetic induction equation on the basis of low Reynolds approximation [15, 16]. The dimensionless magnetic Reynolds number ($Re_m = \sigma\mu_0\nu$) of flow represents the ratio of advection to diffusion in a magnetic field. At $Re_m \ll 1$, advection is relatively unimportant, and a magnetic field thus tends to relax toward a purely diffusive state. This relaxation is determined by boundary conditions rather than flow; consequently, fluid motion has no influence on magnetic field distribution.

In this work, a simulation of magneto-convective flow was carried out inside a square cavity. The purpose of the current study was to numerically solve full magnetic induction, continuum, momentum, and energy equations to investigate the effect of the

magnetic Reynolds number on the magnetic field distribution and temperature and velocity profiles.

2. Basic Equations

The schematic of the examined enclosure is depicted in Fig. 1. The cavity is differentially heated; the left and right walls are isothermal at T_H and T_C , respectively ($T_H > T_C$); and the horizontal walls are adiabatic. A magnetic field was applied in the X direction.

The governing equations used in this research are based on the conservation laws of mass, momentum, energy, and induction equations. The governing equations are provided below:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0 \tag{1}$$

$$\rho(u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y}) = -\frac{\partial P}{\partial x} + \mu(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2}) - \frac{1}{\mu_0}(\frac{\partial B_y}{\partial x} - \frac{\partial B_x}{\partial y})B_y \tag{2}$$

$$\rho(u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y}) = -\frac{\partial P}{\partial y} + \mu(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2}) + \rho g \beta (T - T_c) + \frac{1}{\mu_0}(\frac{\partial B_y}{\partial x} - \frac{\partial B_x}{\partial y})B_x \tag{3}$$

$$\rho C_p \left(u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right) = k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) \tag{4}$$

$$u \frac{\partial B_x}{\partial x} + v \frac{\partial B_x}{\partial y} = B_x \frac{\partial u}{\partial x} + B_y \frac{\partial u}{\partial y} + \frac{1}{\sigma \mu_0} \left(\frac{\partial^2 B_x}{\partial x^2} + \frac{\partial^2 B_x}{\partial y^2} \right) \tag{5}$$

$$u \frac{\partial B_y}{\partial x} + v \frac{\partial B_y}{\partial y} = B_x \frac{\partial v}{\partial x} + B_y \frac{\partial v}{\partial y} + \frac{1}{\sigma \mu_0} \left(\frac{\partial^2 B_y}{\partial x^2} + \frac{\partial^2 B_y}{\partial y^2} \right) \tag{6}$$

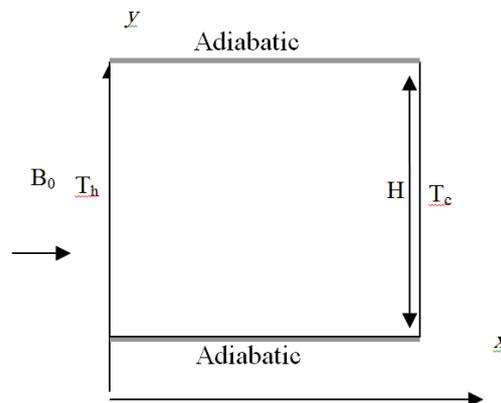


Fig. 1. Geometry and coordinates of cavity configuration with magnetic effect

More details regarding Equations (5) and (6) can be found in [17]. In the equations above, u and v are velocity components, p denotes the pressure, T is the temperature, ρ represents the density, g is the gravitational acceleration, μ is the viscosity, β is the coefficient of thermal expansion, B_x stands for the magnetic field in the X direction, and B_y represents the magnetic field in the Y direction. Moreover, k pertains to the thermal conductivity, C_p is the specific heat capacity, σ is the electrical conductivity, and μ_0 denotes the magnetic permeability.

3. The dimensionless variables included in the analysis are defined thus:

$$X = \frac{x}{H}, Y = \frac{y}{H}, U = \frac{\rho C_p u H}{k}, V = \frac{\rho C_p v H}{k},$$

$$P = \frac{\rho C_p p^2 H^2}{k^2}, \theta = \frac{T - T_c}{T_h - T_c}, Re_m = \sigma \mu_0 \nu,$$

$$B_x^* = \frac{B_x}{B_0}, B_y^* = \frac{B_y}{B_0}, Pr = \frac{k}{\rho c_p} \tag{7}$$

$$Ra = \frac{c_p g \beta (T_h - T_c) H^3}{k \mu}, Ha = B_0 H \sqrt{\frac{\sigma}{\mu}}$$

On the basis of these variables, the governing equations are expressed in dimensionless form as follows:

$$\frac{\partial U}{\partial X} + \frac{\partial V}{\partial Y} = 0 \tag{8}$$

$$U \frac{\partial U}{\partial X} + V \frac{\partial U}{\partial Y} = -\frac{\partial P}{\partial X} + Pr \left(\frac{\partial^2 U}{\partial X^2} + \frac{\partial^2 U}{\partial Y^2} \right) - Ha^2 \frac{Pr^2}{Re_m} B_y^* \left(\frac{\partial B_y^*}{\partial X} - \frac{\partial B_x^*}{\partial Y} \right) \tag{9}$$

$$U \frac{\partial V}{\partial X} + V \frac{\partial V}{\partial Y} = -\frac{\partial P}{\partial Y} + Pr \left(\frac{\partial^2 V}{\partial X^2} + \frac{\partial^2 V}{\partial Y^2} \right) + Ra Pr \theta + Ha^2 \frac{Pr^2}{Re_m} B_x^* \left(\frac{\partial B_y^*}{\partial X} - \frac{\partial B_x^*}{\partial Y} \right) \tag{10}$$

$$U \frac{\partial \theta}{\partial X} + V \frac{\partial \theta}{\partial Y} = \frac{\partial^2 \theta}{\partial X^2} + \frac{\partial^2 \theta}{\partial Y^2} \tag{11}$$

$$U \frac{\partial B_x^*}{\partial X} + V \frac{\partial B_x^*}{\partial Y} = B_x^* \frac{\partial U}{\partial X} + B_y^* \frac{\partial U}{\partial Y} + \frac{Pr}{Re_m} \left(\frac{\partial^2 B_x^*}{\partial X^2} + \frac{\partial^2 B_x^*}{\partial Y^2} \right) \tag{12}$$

$$U \frac{\partial B_y^*}{\partial X} + V \frac{\partial B_y^*}{\partial Y} = B_x^* \frac{\partial V}{\partial X} + B_y^* \frac{\partial V}{\partial Y} + \frac{Pr}{Re_m} \left(\frac{\partial^2 B_y^*}{\partial X^2} + \frac{\partial^2 B_y^*}{\partial Y^2} \right) \tag{13}$$

To calculate the magnetic field instead of solving Equations (12) and (13), we can use the magnetic potential vector (\vec{A}) as follows:

$$\vec{B} = \nabla \times \vec{A} \tag{14}$$

$$B_x = \frac{\partial A_z}{\partial Y}, B_y = -\frac{\partial A_z}{\partial X} = 0, B_z = \frac{\partial A_z}{\partial Z} = 0 \tag{15}$$

Thus, Equations (12) and (13) are converted into Equation (15) in the following manner:

$$U \frac{\partial A}{\partial X} + V \frac{\partial A}{\partial Y} = \frac{Pr}{Re_m} \left(\frac{\partial^2 A}{\partial X^2} + \frac{\partial^2 A}{\partial Y^2} \right) \tag{16}$$

The boundary conditions are

$$A = Y, B_x = 1, \frac{\partial B_y}{\partial X} = 0 \text{ at } X = 0 \tag{17}$$

$$A = Y, B_x = 1, \frac{\partial B_y}{\partial X} = 0 \text{ at } X = 1 \tag{18}$$

$$A = 0, B_y = 0, \frac{\partial B_x}{\partial Y} = 0 \text{ at } Y = 0 \tag{19}$$

$$A = 1, B_y = 0, \frac{\partial B_x}{\partial Y} = 0 \text{ at } Y = 1 \tag{20}$$

$$U \ \& \ V = 0 \text{ at all walls} \tag{21}$$

$$(X = 0, X = 1, Y = 0, Y = 1) \theta(0, Y) = 1, \theta(1, Y) = 0, \frac{\partial \theta}{\partial Y} \Big|_{Y=0} = 0, \frac{\partial \theta}{\partial Y} \Big|_{Y=1} = 0 \tag{22}$$

4. Numerical Procedures

The governing equations associated with the boundary conditions were solved numerically using the control volume-based finite volume method. The hybrid scheme, which is a combination of the central difference and upwind schemes, was used to discretize convection terms. A staggered grid system, in which velocity components are stored midway between scalar storage locations, was also employed. The well-known revised semi-implicit method for pressure linked equations algorithm was adopted in coupling the velocity field and pressure in the momentum equations. The solution of the fully coupled discretized equations was iteratively obtained using the tri-diagonal matrix algorithm. The values of residuals were checked on the basis of physical variables, such

as temperature, pressure, and velocity. Convergence was considered achieved when the summation of residuals was less than 10^{-4} , as was the case for most of the dependent variables. Simulations were conducted with cell numbers ranging from 31×31 to 61×61 at a Rayleigh number (Ra) of 10^5 . Fig. 2 shows the grid distributions of the cavity.

5. Results and Discussion

To verify the accuracy of the numerical technique used to solve the problem considered in this work, a simulation of magneto-convective flow was carried out; the examined structure was a square enclosure with a horizontal temperature gradient and was simulated in the presence of the magnetic field reported by Sarris [18]. Fig. 3 plots the streamlines and isotherms of the solution derived in the current research and the results obtained by Sarris for $Ra=7 \times 10^5$, $Ha=100$, and $Pr=0.7$.

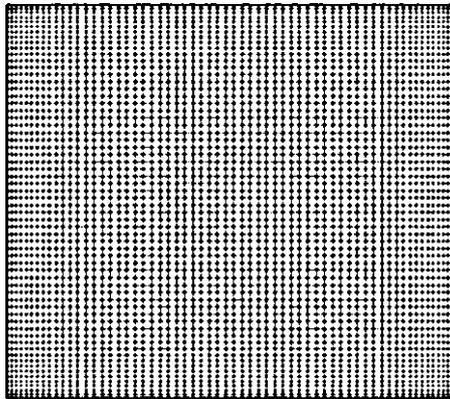


Fig. 2. Grid distributions of the cavity

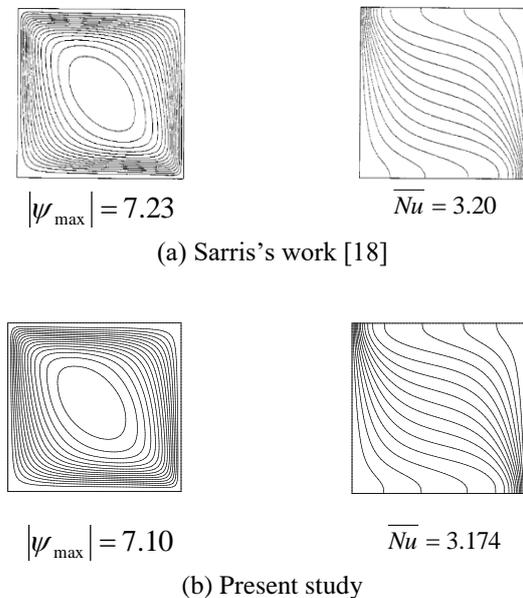


Fig. 3. Comparison of isotherms and streamlines in the present work and Sarris's study at $Ra=7 \times 10^5$ and $Ha=100$

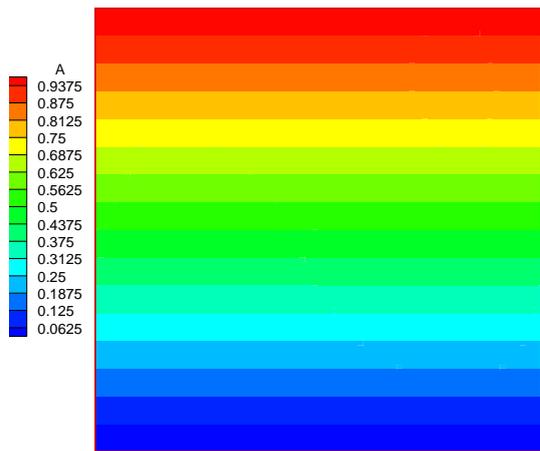
In the comparison of results, the magnetic field induced by the motion of the electrically conducting fluid was disregarded. Our findings showed good agreement with those of Sarris. The relative errors in the average Nusselt number and the maximum absolute value of the stream function of Sarris work [18] and the present model were 0.8% and 1.8%, respectively. In the current research, three magnetic Reynolds numbers, namely, 10^{-1} , 10^{-3} , and 10^{-5} , were adopted. The fluid used was liquid sodium with $Pr=0.01$ and $Ra=10^5$.

Fig. 4 shows the contours of A , B_x , and B_y at $Re_m=10^{-5}$ and $Ha=80$. As expected, because the magnetic field was very small, the magnetic potential from the left to the right sides of the cavity diffused, and its quantity remained identical to that introduced into the boundary conditions. Fluid flow did not affect the applied magnetic field. Variables B_x and B_y were calculated on the basis of the gradient of A ; thus, they were constant in the entire cavity. Specifically, B_x was approximately equal to 1, and B_y was equal to 0. The magnetic field in the cavity was preserved.

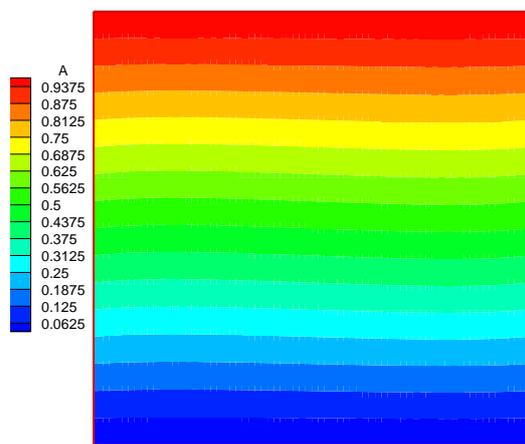
Fig. 5 illustrates the contours of A , B_x , and B_y at $Re_m=10^{-3}$ and $Ha=80$. Given that the Re_m was small, the ratio of the diffusion of the magnetic potential to its advection was large. Consequently, the contours of A were approximately horizontal, and B_x and B_y in the cavity were constant. The maximum deviation of B_x from 1 was 2.73%, and the maximum deviation of B_y from 0 was 9.19%.

The contours of A , B_x , and B_y at $Re_m=10^{-1}$ and $Ha=80$ are presented in Fig. 6. The advection of the magnetic potential was greater than its diffusion, thereby generating horizontal contours. The magnetic field was unequal to the applied magnetic field and was inconstant in the cavity. The magnetic field in the X direction (B_x) therefore varied from 0.55 to 6.6, and the magnetic field in the Y direction (B_y) varied from -1.164 to 4.05. The concentration of the B_x contours near the horizontal walls was greater than that near the vertical walls. The magnetic field was stronger near the adiabatic walls. Furthermore, the concentration of the B_y contours near the vertical walls was higher than that near the horizontal walls.

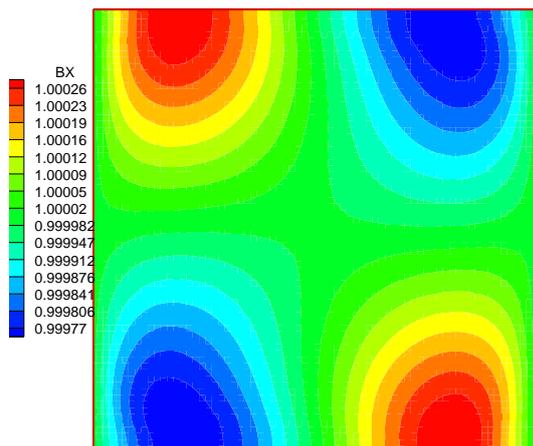
The advection of the magnetic potential was greater than its diffusion, thereby producing non-horizontal contours. The magnetic field was unequal to the applied magnetic field and was inconstant in the cavity. As a result, the magnetic field in the X direction (B_x) varied from 0.55 to 6.6, and the magnetic field in the Y direction (B_y) varied from -1.164 to 4.05. The concentration of the B_x contours near the horizontal walls was higher than that near the vertical walls. The magnetic field was stronger near the adiabatic walls. The concentration of the B_y contours near the vertical walls was greater than that near the horizontal walls.



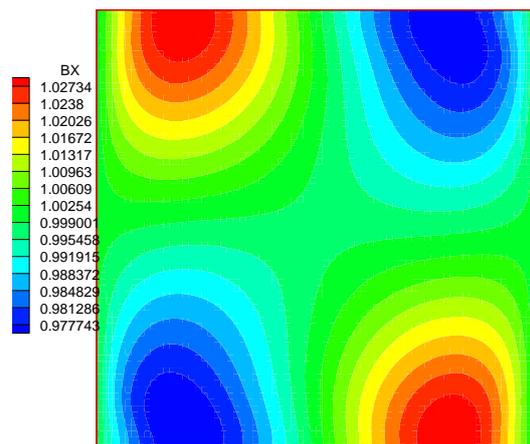
(a)



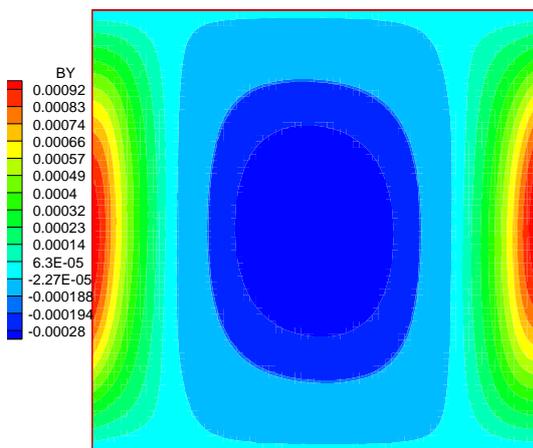
(a)



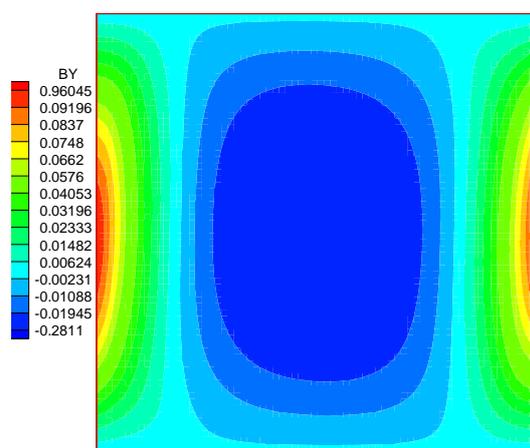
(b)



(b)



(c)



(c)

Fig. 4. Contours of variables: (a) contours of magnetic potential A, (b) contours of magnetic field in the X direction (B_x), and (c) contours of magnetic field in the Y direction (B_y) at $Re_m=10^{-5}$ and $Ha=80$

Fig. 5. Contours of variables: (a) contours of magnetic potential A, (b) contours of magnetic field in the x direction (B_x), and (c) contours of magnetic field in the Y direction (B_y) at $Re_m=10^{-3}$ and $Ha=80$

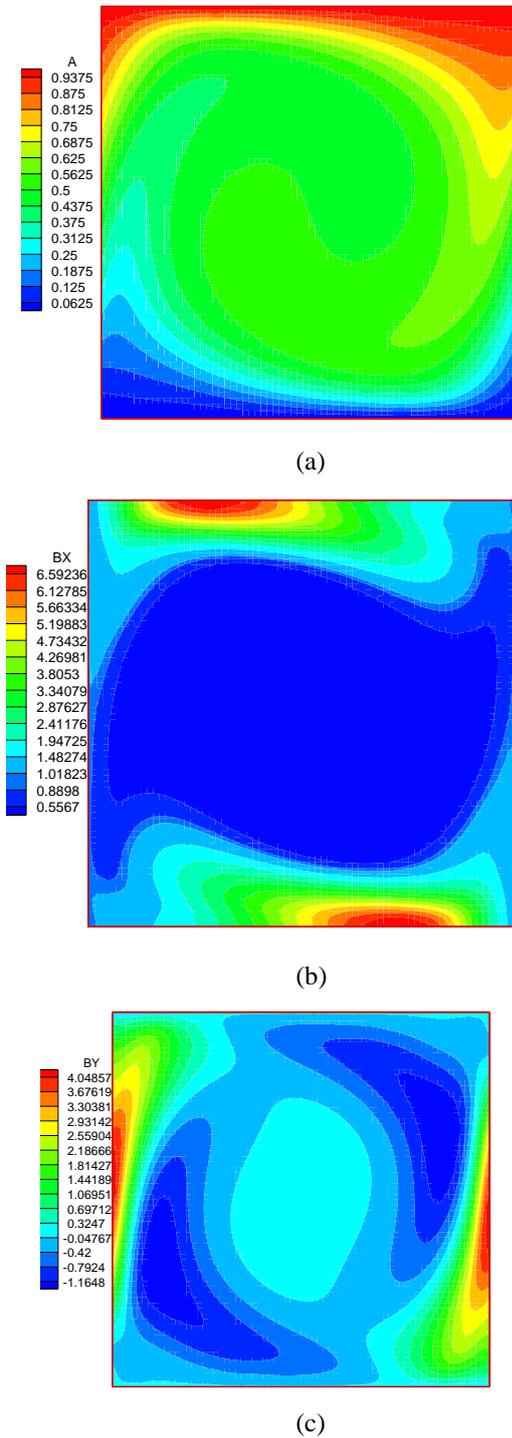


Fig. 6. Contours of variables: (a) contours of magnetic potential A, (b) contours of magnetic field in the X direction (B_x), and (c) contours of magnetic field in the Y direction (B_y) at $Re_m=10^{-1}$ and $Ha=80$

Fig. 7 presents the mid-height velocity profiles at $Ra=10^5$, $Ha=80$, and different magnetic Reynolds numbers. At $Re_m=0.1$, the maximum velocity near the hot wall was higher than those at $Re_m=10^{-3}$ and $Re_m=10^{-5}$. The magnetic field near the hot wall was weaker than that near the adiabatic walls (Fig. 6(b)); thus, the Lorentz force was very low in these regions.

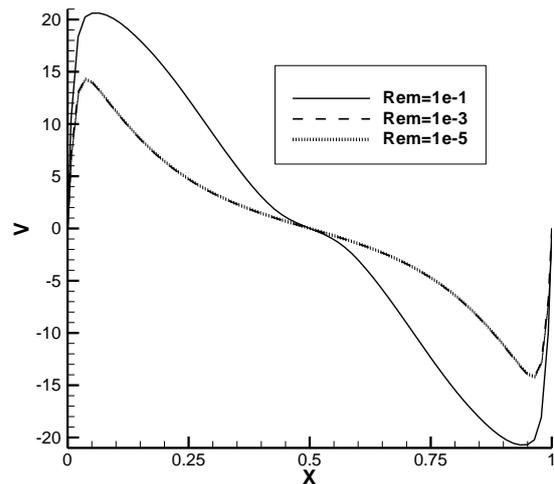


Fig. 7. Velocity profile at $Ra=10^5$, $Ha=80$, and different magnetic Reynolds numbers

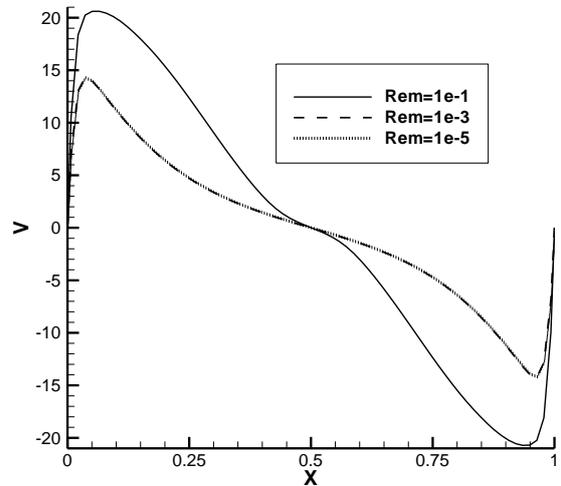


Fig. 8. Temperature velocity profile at $Ra=10^5$, $Ha=80$, and different magnetic Reynolds numbers

Fig. 8 illustrates the mid-height temperature profiles at $Ra=10^5$, $Ha=80$, and different magnetic Reynolds numbers. At $Re_m=0.1$, the temperature gradient and the slope of the temperature profile near the hot wall were higher than those at $Re_m=10^{-3}$ and $Re_m=10^{-5}$.

6. Conclusion

This study analyzed the steady, laminar, and magneto-convective flow of a viscous fluid in a cavity. Temperature gradient was applied on the two opposing regular walls of the enclosure, whereas adiabatic conditions were maintained in the other walls. A magnetic field was applied in the X direction. Magnetic Reynolds numbers of 10^{-1} , 10^{-3} , and 10^{-5} were adopted, and the fluid used was liquid sodium with $Pr=0.01$ and $Ra=10^5$.

As the Re_m increased, the advection of the magnetic potential increased, and variations in the

magnetic field were augmented because fluid flow affected magnetic distribution. The induced magnetic field was negligible, and the magnetic field in the entire cavity was equal to the applied external magnetic field.

Nomenclature

B_0	Magnitude of magnetic field
B_x	Magnitude of magnetic field in the X direction
B_y	Magnitude of magnetic field in the Y direction
g	Acceleration due to gravity
C_p	Heat capacity
A	Magnetic potential vector
H	Height of enclosure
H_a	Hartmann number
Ra	Rayleigh number
Re_m	Magnetic Reynolds number
P	Pressure
k	Thermal conductivity
T	Temperature
u, v	Velocity components
β	Coefficient of thermal expansion
α	Thermal diffusivity
ν	Dynamic viscosity
ρ	Density
σ	Electrical conductivity
θ	Dimensionless temperature
μ_0	Magnetic permeability

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